

- 2.10** A square yellow-brass bar must not stretch more than 2.5 mm when it is subjected to a tensile load. Knowing that  $E = 105$  GPa and that the allowable tensile strength is 180 MPa, determine (a) the maximum allowable length of the bar, (b) the required dimensions of the cross section if the tensile load is 40 kN.
- 2.11** A 4-m-long steel rod must not stretch more than 3 mm and the normal stress must not exceed 150 MPa when the rod is subjected to a 10-kN axial load. Knowing that  $E = 200$  GPa, determine the required diameter of the rod.
- 2.12** A nylon thread is to be subjected to a 10-N tension. Knowing that  $E = 3.2$  GPa, that the maximum allowable normal stress is 40 MPa, and that the length of the thread must not increase by more than 1%, determine the required diameter of the thread.
- 2.13** The 4-mm-diameter cable  $BC$  is made of a steel with  $E = 200$  GPa. Knowing that the maximum stress in the cable must not exceed 190 MPa and that the elongation of the cable must not exceed 6 mm, find the maximum load  $P$  that can be applied as shown.
- 2.14** The aluminum rod  $ABC$  ( $E = 10.1 \times 10^6$  psi), which consists of two cylindrical portions  $AB$  and  $BC$ , is to be replaced with a cylindrical steel rod  $DE$  ( $E = 29 \times 10^6$  psi) of the same overall length. Determine the minimum required diameter  $d$  of the steel rod if its vertical deformation is not to exceed the deformation of the aluminum rod under the same load and if the allowable stress in the steel rod is not to exceed 24 ksi.

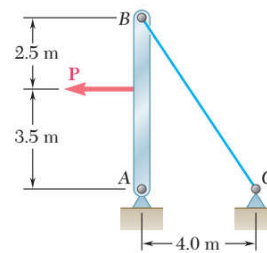


Fig. P2.13

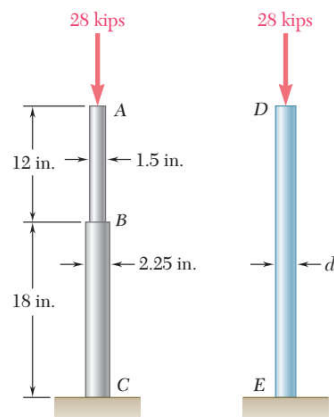


Fig. P2.14

- 2.15** A 4-ft section of aluminum pipe of cross-sectional area  $1.75$  in<sup>2</sup> rests on a fixed support at  $A$ . The  $\frac{5}{8}$ -in.-diameter steel rod  $BC$  hangs from a rigid bar that rests on the top of the pipe at  $B$ . Knowing that the modulus of elasticity is  $29 \times 10^6$  psi for steel and  $10.4 \times 10^6$  psi for aluminum, determine the deflection of point  $C$  when a 15-kip force is applied at  $C$ .

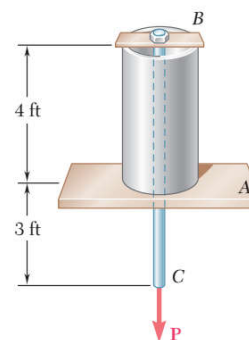


Fig. P2.15

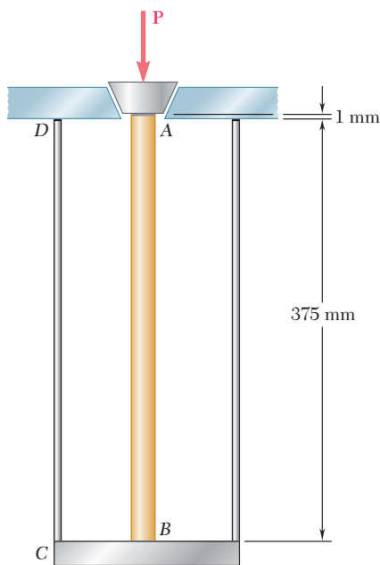


Fig. P2.16

**2.16** The brass tube  $AB$  ( $E = 105$  GPa) has a cross-sectional area of  $140 \text{ mm}^2$  and is fitted with a plug at  $A$ . The tube is attached at  $B$  to a rigid plate that is itself attached at  $C$  to the bottom of an aluminum cylinder ( $E = 72$  GPa) with a cross-sectional area of  $250 \text{ mm}^2$ . The cylinder is then hung from a support at  $D$ . In order to close the cylinder, the plug must move down through  $1 \text{ mm}$ . Determine the force  $\mathbf{P}$  that must be applied to the cylinder.

**2.17** A 250-mm-long aluminum tube ( $E = 70$  GPa) of 36-mm outer diameter and 28-mm inner diameter can be closed at both ends by means of single-threaded screw-on covers of 1.5-mm pitch. With one cover screwed on tight, a solid brass rod ( $E = 105$  GPa) of 25-mm diameter is placed inside the tube and the second cover is screwed on. Since the rod is slightly longer than the tube, it is observed that the cover must be forced against the rod by rotating it one-quarter of a turn before it can be tightly closed. Determine (a) the average normal stress in the tube and in the rod, (b) the deformations of the tube and of the rod.

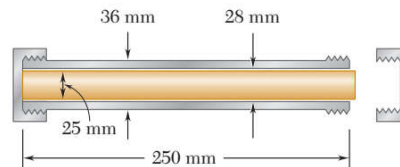


Fig. P2.17

**2.18** The specimen shown is made from a 1-in.-diameter cylindrical steel rod with two 1.5-in.-outer-diameter sleeves bonded to the rod as shown. Knowing that  $E = 29 \times 10^6$  psi, determine (a) the load  $\mathbf{P}$  so that the total deformation is  $0.002 \text{ in.}$ , (b) the corresponding deformation of the central portion  $BC$ .

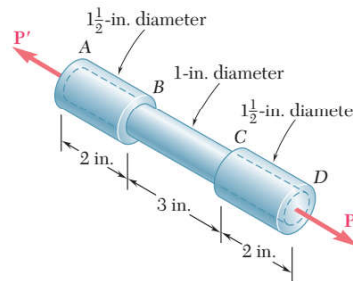


Fig. P2.18

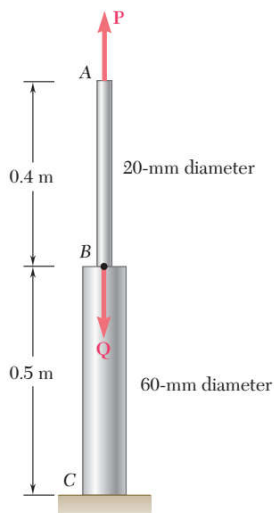
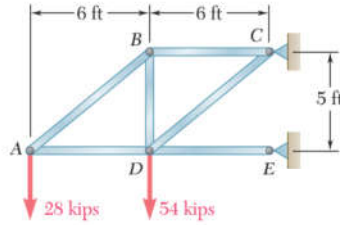


Fig. P2.19 and P2.20

**2.19** Both portions of the rod  $ABC$  are made of an aluminum for which  $E = 70$  GPa. Knowing that the magnitude of  $\mathbf{P}$  is  $4 \text{ kN}$ , determine (a) the value of  $\mathbf{Q}$  so that the deflection at  $A$  is zero, (b) the corresponding deflection of  $B$ .

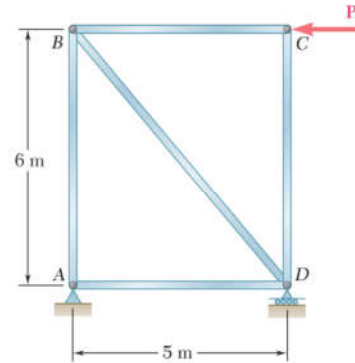
**2.20** The rod  $ABC$  is made of an aluminum for which  $E = 70$  GPa. Knowing that  $P = 6 \text{ kN}$  and  $Q = 42 \text{ kN}$ , determine the deflection of (a) point  $A$ , (b) point  $B$ .

- 2.21** Members  $AB$  and  $BC$  are made of steel ( $E = 29 \times 10^6$  psi) with cross-sectional areas of  $0.80 \text{ in}^2$  and  $0.64 \text{ in}^2$ , respectively. For the loading shown, determine the elongation of (a) member  $AB$ , (b) member  $BC$ .



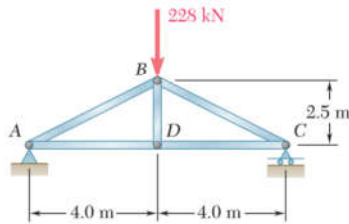
**Fig. P2.21**

- 2.22** The steel frame ( $E = 200 \text{ GPa}$ ) shown has a diagonal brace  $BD$  with an area of  $1920 \text{ mm}^2$ . Determine the largest allowable load  $P$  if the change in length of member  $BD$  is not to exceed  $1.6 \text{ mm}$ .



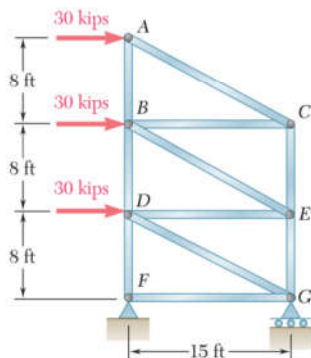
**Fig. P2.22**

- 2.23** For the steel truss ( $E = 200 \text{ GPa}$ ) and loading shown, determine the deformations of members  $AB$  and  $AD$ , knowing that their cross-sectional areas are  $2400 \text{ mm}^2$  and  $1800 \text{ mm}^2$ , respectively.



**Fig. P2.23**

- 2.24** For the steel truss ( $E = 29 \times 10^6$  psi) and loading shown, determine the deformations of members  $BD$  and  $DE$ , knowing that their cross-sectional areas are  $2 \text{ in}^2$  and  $3 \text{ in}^2$ , respectively.



**Fig. P2.24**

- 2.25** Each of the links  $AB$  and  $CD$  is made of aluminum ( $E = 10.9 \times 10^6$  psi) and has a cross-sectional area of  $0.2 \text{ in}^2$ . Knowing that they support the rigid member  $BC$ , determine the deflection of point  $E$ .

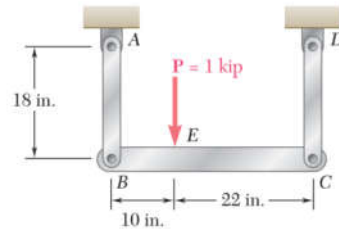


Fig. P2.25

- 2.26** The length of the  $\frac{3}{32}$ -in.-diameter steel wire  $CD$  has been adjusted so that with no load applied, a gap of  $\frac{1}{16}$  in. exists between the end  $B$  of the rigid beam  $ACB$  and a contact point  $E$ . Knowing that  $E = 29 \times 10^6$  psi, determine where a 50-lb block should be placed on the beam in order to cause contact between  $B$  and  $E$ .

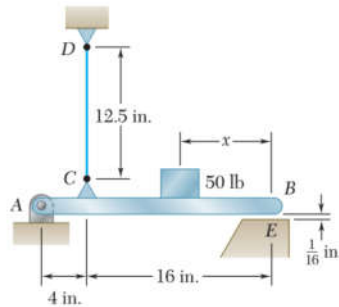


Fig. P2.26

- 2.27** Link  $BD$  is made of brass ( $E = 105 \text{ GPa}$ ) and has a cross-sectional area of  $240 \text{ mm}^2$ . Link  $CE$  is made of aluminum ( $E = 72 \text{ GPa}$ ) and has a cross-sectional area of  $300 \text{ mm}^2$ . Knowing that they support rigid member  $ABC$ , determine the maximum force  $\mathbf{P}$  that can be applied vertically at point  $A$  if the deflection of  $A$  is not to exceed  $0.35 \text{ mm}$ .

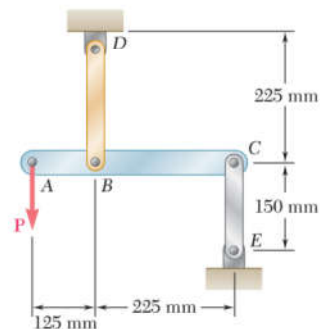


Fig. P2.27

- 2.28** Each of the four vertical links connecting the two rigid horizontal links is made of aluminum ( $E = 70 \text{ GPa}$ ) and has a uniform rectangular cross section of  $10 \times 40 \text{ mm}$ . For the loading shown, determine the deflection of (a) point E, (b) point F, (c) point G.

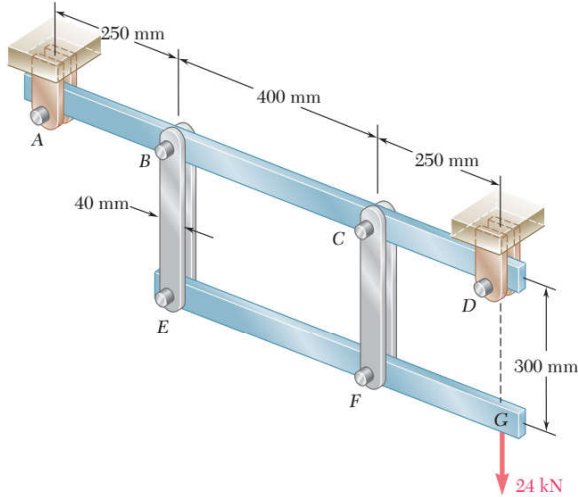


Fig. P2.28

- 2.29** The vertical load  $P$  is applied at the center A of the upper section of a homogeneous frustum of a circular cone of height  $h$ , minimum radius  $a$ , and maximum radius  $b$ . Denoting by  $E$  the modulus of elasticity of the material and neglecting the effect of its weight, determine the deflection of point A.

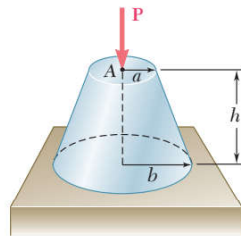


Fig. P2.29

- 2.30** A homogenous cable of length  $L$  and uniform cross section is suspended from one end. (a) Denoting by  $\rho$  the density (mass per unit volume) of the cable and by  $E$  its modulus of elasticity, determine the elongation of the cable due to its own weight. (b) Show that the same elongation would be obtained if the cable were horizontal and if a force equal to half of its weight were applied at each end.
- 2.31** The volume of a tensile specimen is essentially constant while plastic deformation occurs. If the initial diameter of the specimen is  $d_1$ , show that when the diameter is  $d$ , the true strain is  $\epsilon_t = 2 \ln(d_1/d)$ .
- 2.32** Denoting by  $\epsilon$  the “engineering strain” in a tensile specimen, show that the true strain is  $\epsilon_t = \ln(1 + \epsilon)$ .

# PROBLEMS

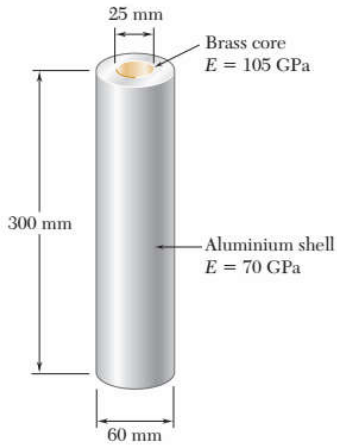


Fig. P2.33 and P2.34

**2.33** An axial force of 200 kN is applied to the assembly shown by means of rigid end plates. Determine (a) the normal stress in the aluminum shell, (b) the corresponding deformation of the assembly.

**2.34** The length of the assembly shown decreases by 0.40 mm when an axial force is applied by means of rigid end plates. Determine (a) the magnitude of the applied force, (b) the corresponding stress in the brass core.

**2.35** A 4-ft concrete post is reinforced with four steel bars, each with a  $\frac{3}{4}$ -in. diameter. Knowing that  $E_s = 29 \times 10^6$  psi and  $E_c = 3.6 \times 10^6$  psi, determine the normal stresses in the steel and in the concrete when a 150-kip axial centric force  $P$  is applied to the post.

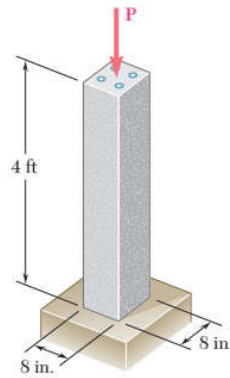


Fig. P2.35

**2.36** A 250-mm bar of  $150 \times 30$ -mm rectangular cross section consists of two aluminum layers, 5 mm thick, brazed to a center brass layer of the same thickness. If it is subjected to centric forces of magnitude  $P = 30$  kN, and knowing that  $E_a = 70$  GPa and  $E_b = 105$  GPa, determine the normal stress (a) in the aluminum layers, (b) in the brass layer.

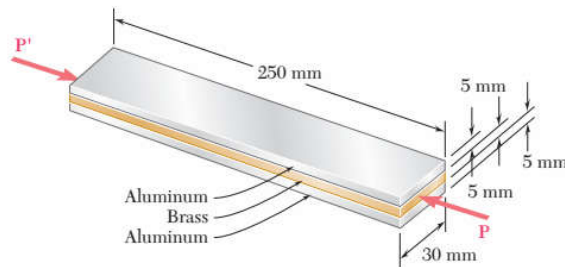


Fig. P2.36

**2.37** Determine the deformation of the composite bar of Prob. 2.36 if it is subjected to centric forces of magnitude  $P = 45$  kN.



**2.38** Compressive centric forces of 40 kips are applied at both ends of the assembly shown by means of rigid end plates. Knowing that  $E_s = 29 \times 10^6$  psi and  $E_a = 10.1 \times 10^6$  psi, determine (a) the normal stresses in the steel core and the aluminum shell, (b) the deformation of the assembly.

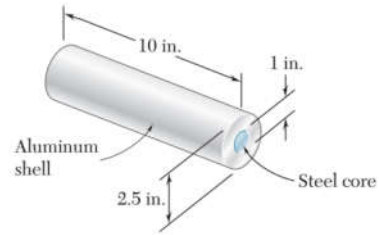


Fig. P2.38

**2.39** Three wires are used to suspend the plate shown. Aluminum wires of  $\frac{1}{8}$ -in. diameter are used at A and B while a steel wire of  $\frac{1}{12}$ -in. diameter is used at C. Knowing that the allowable stress for aluminum ( $E_a = 10.4 \times 10^6$  psi) is 14 ksi and that the allowable stress for steel ( $E_s = 29 \times 10^6$  psi) is 18 ksi, determine the maximum load **P** that can be applied.

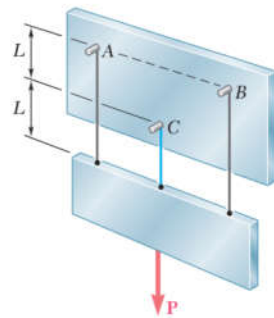


Fig. P2.39

**2.40** A polystyrene rod consisting of two cylindrical portions AB and BC is restrained at both ends and supports two 6-kip loads as shown. Knowing that  $E = 0.45 \times 10^6$  psi, determine (a) the reactions at A and C, (b) the normal stress in each portion of the rod.

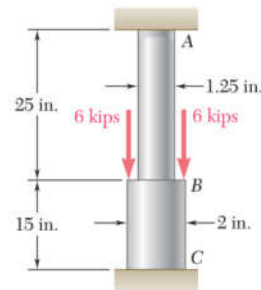


Fig. P2.40

**2.41** Two cylindrical rods, one of steel and the other of brass, are joined at C and restrained by rigid supports at A and E. For the loading shown and knowing that  $E_s = 200$  GPa and  $E_b = 105$  GPa, determine (a) the reactions at A and E, (b) the deflection of point C.

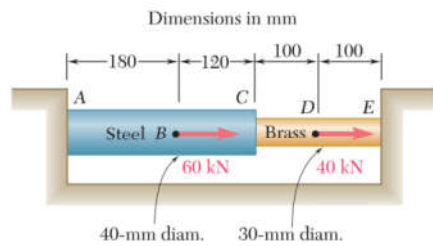


Fig. P2.41

**2.42** Solve Prob. 2.41, assuming that rod AC is made of brass and rod CE is made of steel.

**2.43** The rigid bar ABCD is suspended from four identical wires. Determine the tension in each wire caused by the load **P** shown.

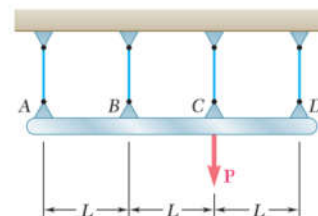


Fig. P2.43

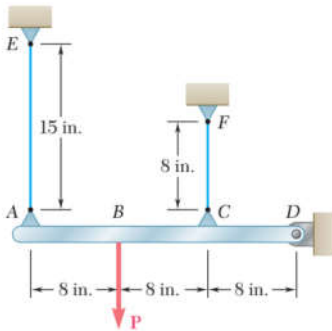


Fig. P2.44

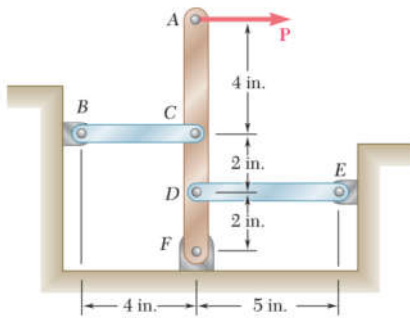


Fig. P2.46

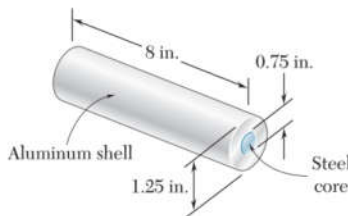


Fig. P2.48

**2.44** The rigid bar  $AD$  is supported by two steel wires of  $\frac{1}{16}$ -in. diameter ( $E = 29 \times 10^6$  psi) and a pin and bracket at  $D$ . Knowing that the wires were initially taut, determine (a) the additional tension in each wire when a 120-lb load  $P$  is applied at  $B$ , (b) the corresponding deflection of point  $B$ .

**2.45** The steel rods  $BE$  and  $CD$  each have a 16-mm diameter ( $E = 200$  GPa); the ends of the rods are single-threaded with a pitch of 2.5 mm. Knowing that after being snugly fitted, the nut at  $C$  is tightened one full turn, determine (a) the tension in rod  $CD$ , (b) the deflection of point  $C$  of the rigid member  $ABC$ .

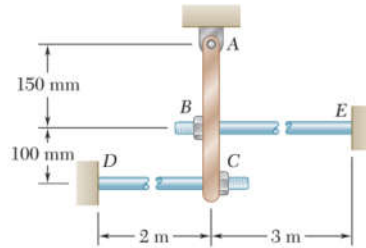


Fig. P2.45

**2.46** Links  $BC$  and  $DE$  are both made of steel ( $E = 29 \times 10^6$  psi) and are  $\frac{1}{2}$  in. wide and  $\frac{1}{4}$  in. thick. Determine (a) the force in each link when a 600-lb force  $P$  is applied to the rigid member  $AF$  shown, (b) the corresponding deflection of point  $A$ .

**2.47** The concrete post ( $E_c = 3.6 \times 10^6$  psi and  $\alpha_c = 5.5 \times 10^{-6}/^\circ\text{F}$ ) is reinforced with six steel bars, each of  $\frac{7}{8}$ -in diameter ( $E_s = 29 \times 10^6$  psi and  $\alpha_s = 6.5 \times 10^{-6}/^\circ\text{F}$ ). Determine the normal stresses induced in the steel and in the concrete by a temperature rise of  $65^\circ\text{F}$ .

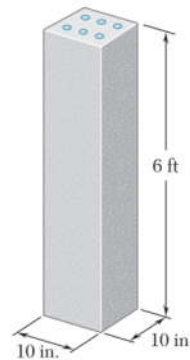


Fig. P2.47

**2.48** The assembly shown consists of an aluminum shell ( $E_a = 10.6 \times 10^6$  psi,  $\alpha_a = 12.9 \times 10^{-6}/^\circ\text{F}$ ) fully bonded to a steel core ( $E_s = 29 \times 10^6$  psi,  $\alpha_s = 6.5 \times 10^{-6}/^\circ\text{F}$ ) and is unstressed. Determine (a) the largest allowable change in temperature if the stress in the aluminum shell is not to exceed 6 ksi, (b) the corresponding change in length of the assembly.



**2.49** The aluminum shell is fully bonded to the brass core and the assembly is unstressed at a temperature of 15°C. Considering only axial deformations, determine the stress in the aluminum when the temperature reaches 195°C.

**2.50** Solve Prob. 2.49, assuming that the core is made of steel ( $E_s = 200 \text{ GPa}$ ,  $\alpha_s = 11.7 \times 10^{-6}/^\circ\text{C}$ ) instead of brass.

**2.51** A rod consisting of two cylindrical portions  $AB$  and  $BC$  is restrained at both ends. Portion  $AB$  is made of steel ( $E_s = 200 \text{ GPa}$ ,  $\alpha_s = 11.7 \times 10^{-6}/^\circ\text{C}$ ) and portion  $BC$  is made of brass ( $E_b = 105 \text{ GPa}$ ,  $\alpha_b = 20.9 \times 10^{-6}/^\circ\text{C}$ ). Knowing that the rod is initially unstressed, determine the compressive force induced in  $ABC$  when there is a temperature rise of 50°C.

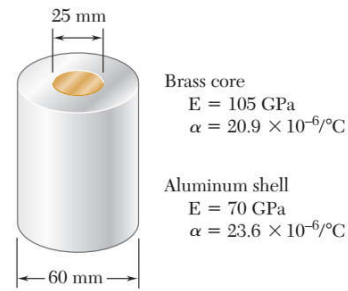


Fig. P2.49

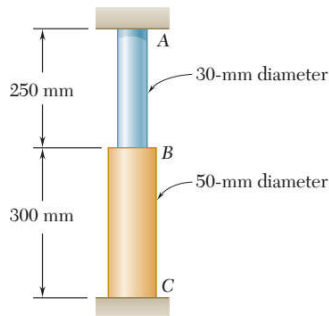


Fig. P2.51

**2.52** A steel railroad track ( $E_s = 200 \text{ GPa}$ ,  $\alpha_s = 11.7 \times 10^{-6}/^\circ\text{C}$ ) was laid out at a temperature of 6°C. Determine the normal stress in the rails when the temperature reaches 48°C, assuming that the rails ( $a$ ) are welded to form a continuous track, ( $b$ ) are 10 m long with 3-mm gaps between them.

**2.53** A rod consisting of two cylindrical portions  $AB$  and  $BC$  is restrained at both ends. Portion  $AB$  is made of steel ( $E_s = 29 \times 10^6 \text{ psi}$ ,  $\alpha_s = 6.5 \times 10^{-6}/^\circ\text{F}$ ) and portion  $BC$  is made of aluminum ( $E_a = 10.4 \times 10^6 \text{ psi}$ ,  $\alpha_a = 13.3 \times 10^{-6}/^\circ\text{F}$ ). Knowing that the rod is initially unstressed, determine ( $a$ ) the normal stresses induced in portions  $AB$  and  $BC$  by a temperature rise of 70°F, ( $b$ ) the corresponding deflection of point  $B$ .

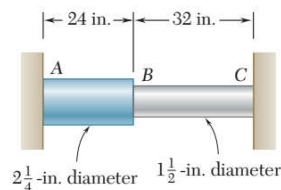
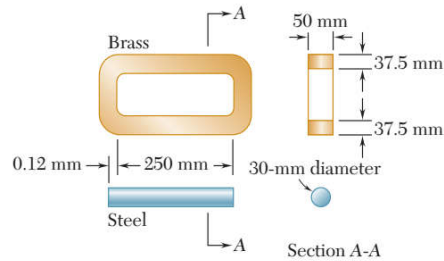


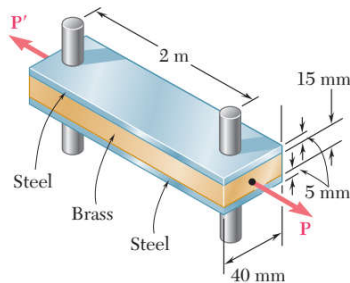
Fig. P2.53

**2.54** Solve Prob. 2.53, assuming that portion  $AB$  of the composite rod is made of aluminum and portion  $BC$  is made of steel.

**2.55** A brass link ( $E_b = 105 \text{ GPa}$ ,  $\alpha_b = 20.9 \times 10^{-6}/^\circ\text{C}$ ) and a steel rod ( $E_s = 200 \text{ GPa}$ ,  $\alpha_s = 11.7 \times 10^{-6}/^\circ\text{C}$ ) have the dimensions shown at a temperature of  $20^\circ\text{C}$ . The steel rod is cooled until it fits freely into the link. The temperature of the whole assembly is then raised to  $45^\circ\text{C}$ . Determine (a) the final normal stress in the steel rod, (b) the final length of the steel rod.



**Fig. P2.55**



**Fig. P2.56**

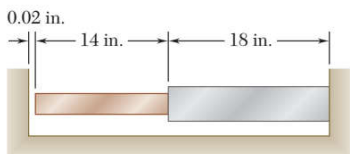
**2.56** Two steel bars ( $E_s = 200 \text{ GPa}$  and  $\alpha_s = 11.7 \times 10^{-6}/^\circ\text{C}$ ) are used to reinforce a brass bar ( $E_b = 105 \text{ GPa}$ ,  $\alpha_b = 20.9 \times 10^{-6}/^\circ\text{C}$ ) that is subjected to a load  $P = 25 \text{ kN}$ . When the steel bars were fabricated, the distance between the centers of the holes that were to fit on the pins was made  $0.5 \text{ mm}$  smaller than the  $2 \text{ m}$  needed. The steel bars were then placed in an oven to increase their length so that they would just fit on the pins. Following fabrication, the temperature in the steel bars dropped back to room temperature. Determine (a) the increase in temperature that was required to fit the steel bars on the pins, (b) the stress in the brass bar after the load is applied to it.

**2.57** Determine the maximum load  $P$  that can be applied to the brass bar of Prob. 2.56 if the allowable stress in the steel bars is  $30 \text{ MPa}$  and the allowable stress in the brass bar is  $25 \text{ MPa}$ .

**2.58** Knowing that a  $0.02\text{-in.}$  gap exists when the temperature is  $75^\circ\text{F}$ , determine (a) the temperature at which the normal stress in the aluminum bar will be equal to  $-11 \text{ ksi}$ , (b) the corresponding exact length of the aluminum bar.

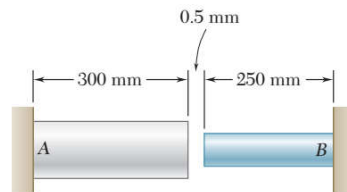
**2.59** Determine (a) the compressive force in the bars shown after a temperature rise of  $180^\circ\text{F}$ , (b) the corresponding change in length of the bronze bar.

**2.60** At room temperature ( $20^\circ\text{C}$ ) a  $0.5\text{-mm}$  gap exists between the ends of the rods shown. At a later time when the temperature has reached  $140^\circ\text{C}$ , determine (a) the normal stress in the aluminum rod, (b) the change in length of the aluminum rod.



Bronze	Aluminum
$A = 2.4 \text{ in}^2$	$A = 2.8 \text{ in}^2$
$E = 15 \times 10^6 \text{ psi}$	$E = 10.6 \times 10^6 \text{ psi}$
$\alpha = 12 \times 10^{-6}/^\circ\text{F}$	$\alpha = 12.9 \times 10^{-6}/^\circ\text{F}$

**Fig. P2.58 and P2.59**



Aluminum	Stainless steel
$A = 2000 \text{ mm}^2$	$A = 800 \text{ mm}^2$
$E = 75 \text{ GPa}$	$E = 190 \text{ GPa}$
$\alpha = 23 \times 10^{-6}/^\circ\text{C}$	$\alpha = 17.3 \times 10^{-6}/^\circ\text{C}$

**Fig. P2.60**